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A Probabilistic LCF Model for Turbo Charger Compressor of a Jet Engine

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We present a probabilistic low cycle fatigue (LCF) Model for turbo charger compressor of a jet engine. This compressor is made of casted aluminum alloy (C355). This model has been implemented using the finite element method in R Language [4].

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1 The Physical Model: The Linear Elasticity Equation

Let us consider a domain $\Omega \subset \mathbb{R}^3$ which represent a deformable medium like a turbo charger compressor. The function $f: \Omega \to \mathbb{R}^3$ represent the external load applied an Ω . Moreover let $u: \Omega \to \mathbb{R}^3$ the displacement field. The boundary of the compressor $\partial\Omega$ is partitioned into a clamped boundary $\partial\Omega_D$ and a Neumann boundary $\partial\Omega_N$. The normal load imposed on the surface boundary is described by the function $g: \partial\Omega \to \mathbb{R}^3$. According to [1], the following equation describes our linear isotropic elasticity model:

$$\begin{cases} \nabla . \sigma(u) + f = 0, & \text{in } \Omega \\ \sigma(u) = \lambda(\nabla . u)\mathcal{I} + \mu(\nabla u + \nabla u^T), & \text{in } \Omega, \\ u = 0, & \text{on } \partial \Omega_D, \\ \sigma(u) . n = g, & \text{on } \partial \Omega_N \end{cases}$$

Here, λ and μ are the Lamé coefficients. The linearized strain rate tensor $\varepsilon(u) : \Omega \to \mathbb{R}^{3,3}$ is defined as: $\varepsilon(u) = \frac{1}{2}(\nabla u + \nabla u^T)$. Numerical solutions of this equation can be computed by a finite element approximation, confer [7].

2 The local and probabilistic model for LCF

We refer in this section to the LCF-model described in the work [2]. The fatigue process is modelled by a continuous time process, where the time is a random variable N describing the number of cycles to failure. The local and probabilistic model for LCF ist given by the cumulative distribution function:

$$F_N(n) = 1 - \exp\left(-\int_0^n \int_{\partial\Omega} \frac{m}{N_{det}} \left(\frac{s}{N_{det}}\right)^{m-1} dA ds\right) = 1 - \exp\left(-J_{sur}(\Omega, u) \cdot n^m\right)$$

where m is the Weibull shape and $J_{sur}(\Omega, u)^{1/m}$ is the Weibull scale parameter which describes the number N of cycles to crack initiation.

3 Finite Element Approximation

In order to compute the probability of failure we need to apply the finite element method to evaluate the following surface integral:

$$J_{sur}(\Omega, u) = \int_{\partial\Omega} \left(\frac{1}{N_{det}(\varepsilon(\nabla u(x)))} \right)^m dA.$$

For more details concerning the computation of N_{det} , we refer to [2].

Denote by \mathcal{N}_h a collection of the boundary faces. The computation of surface integral $J_{sur}(\Omega, u)$ reduces to evaluating integrals over each element in the collection \mathcal{N}_h :

$$\int_{\partial\Omega} \left(\frac{1}{N_{det}(\varepsilon^e(\nabla u(x)))}\right)^m dA \approx \sum_{F \in \mathcal{T}_h} \int_F \left(\frac{1}{N_{det}(\varepsilon^e(\nabla u(x)))}\right)^m dA,$$

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Fig. 1: The probability of failure and lifetime prediction for turbo charger compressor from different views: Red areas show critical regions

where ε^e is the elastic strain tensor.

According to [1] we denote by $T_F : \widehat{F} \to F$ a \mathcal{C}^1 -diffeomorphism mapping the geometric reference face $\widehat{F} \subset \mathbb{R}^2$ to any face F in \mathcal{N}_h . Let $g_F(\widehat{x}) = (J_F(\widehat{x}))^T J_F(\widehat{x})$ be the metric tensor where $J_F(\widehat{x}) = \frac{\partial T_F(\widehat{x})}{\partial \widehat{x}}$ is the Jacobian matrix of the mapping T_F at \widehat{x} . The change of variables $x = T_K(\widehat{x})$ yields :

$$\int_{F} \left(\frac{1}{N_{det}(\varepsilon^{e}(\nabla u(x)))}\right)^{m} dA = \int_{\widehat{F}} \left(\frac{1}{N_{det}(\varepsilon^{e}(\nabla u(T_{F}(\widehat{x}))))}\right)^{m} \sqrt{det(g_{F}(\widehat{x}))} dA$$

By considering a quadrature on \widehat{F} defined by l_q^∂ Gauss points $\{\widehat{\xi_1^\partial}, ..., \widehat{\xi_{l_q^\partial}^\partial}\}$ and l_q^∂ weights $\{\widehat{\omega_1^\partial}, ..., \widehat{\omega_{l_q^\partial}^\partial}\}$ and by setting $\omega_{lF} = \widehat{\omega_l}\sqrt{det(g_F(\widehat{\xi_l}))}$ and $\xi_{lF} = T_F(\widehat{\xi_l})$ we will have:

$$J_{sur}(\Omega, u) \approx \sum_{F \in \mathcal{T}_h} \sum_{l=1}^{l_q} \omega_{lF}[N_{det}(\varepsilon_{\nu}(\nabla u(\xi_{lF}))))]^{-m}.$$

4 Computations and Results

The discretized cost functional J_{sur} in the previous section is computed for a model jet engine radial turbo compressor obtained from [3]. The preprocessing data (coordinates, elements connectivity and displacements) are handled using the free FEA-Software CalculiX. The compressor is discretized using a 20 node hexahedral element with reduced quadratures (he20r) into 9464 elements. J_{sur} is calculated over all 6356 faces (rectangles) on the compressor boundary. We used for this computation a self implemented FEM-tool in R language. The material parameters have been taken from [5]. The Weibull shape parameter is set m = 2, which is a usual value for polycristalline metal. We followed the approach described in [6] to adapt the CMB-parameters to our compressor model because the size of specimens have a non-negligible influence on crack-initiation life.

Figure 1 shows the probability of failure of turbo charger compressor and the computed local crack formation intensity $\left(\frac{1}{N_{det}}\right)^m$ over all faces on the boundary of the compressor.

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